# An Improved Model of Cryogenic Propellant Stratification in a Rotating, Reduced Gravity Environment

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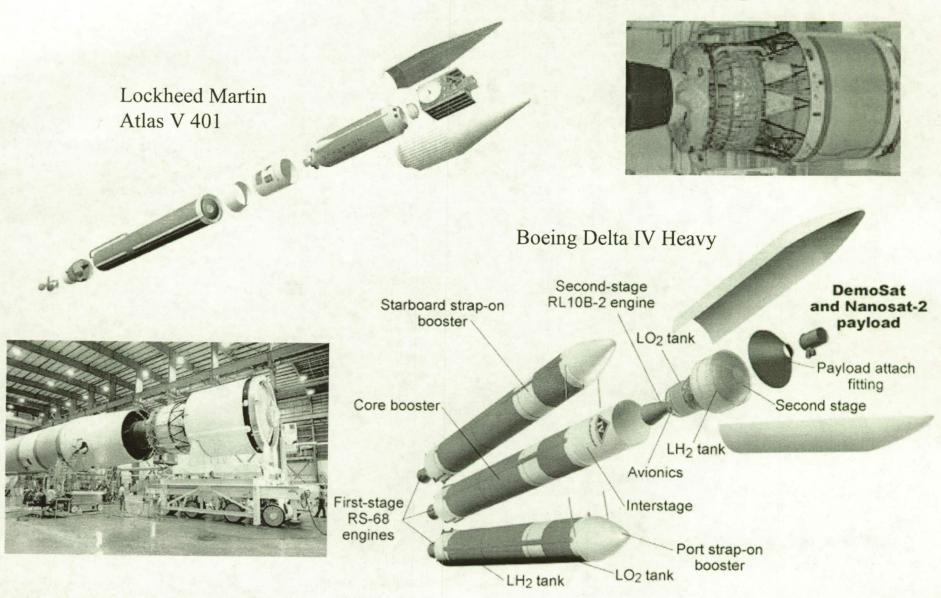
Expendable Launch Vehicle / Mission Analysis
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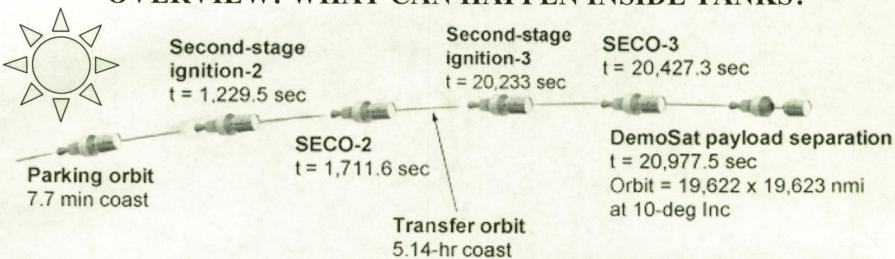
- Project Overview
- Analytical Modeling
- Current Work
- Concluding Remarks
- Future Work

## **OVERVIEW: UPPER STAGE MODELING**

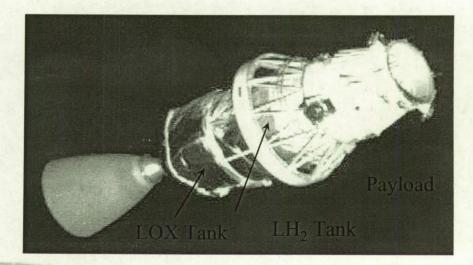


http://www.boeing.com/defense-space/space/delta/delta4/d4h\_demo/book04.html

## **OVERVIEW: WHAT CAN HAPPEN INSIDE TANKS?**



- Stage exposed to solar heating
- Propellants (LH<sub>2</sub> and LOX) may thermally stratify
- Propellants may boil
- Slosh events during maneuvers
- Upper stage must re-start at conclusion of coast phase for insertion

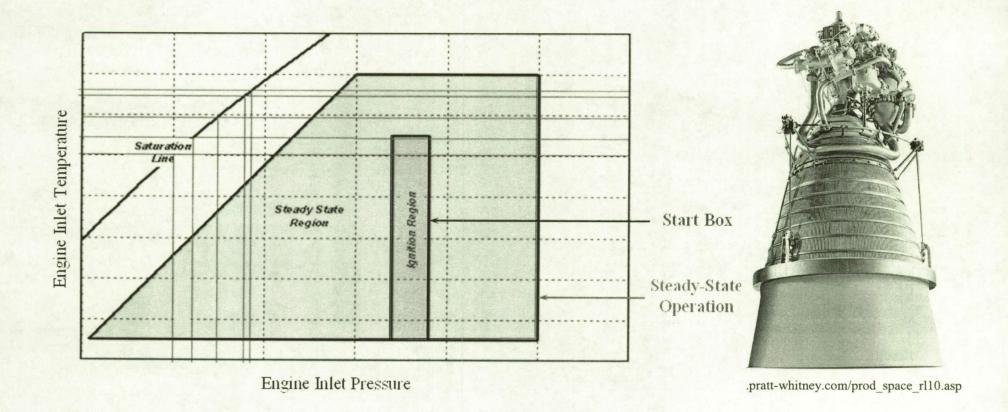


http://www.boeing.com/defense-space/space/delta/delta4/d4h\_demo/book14.html

XSS-10 view of Delta II rocket: An Air Force Research Laboratory XSS-10 micro-satellite uses its onboard camera system to view the second stage of the Boeing Delta II rocket during mission operations Jan. 30. (Photo courtesy of Boeing.), http://www.globalsecurity.org/space/systems/xss.htm

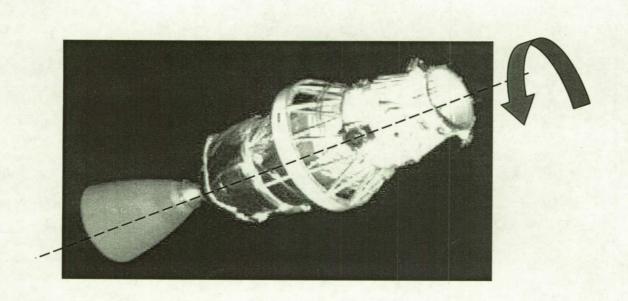
#### **OVERVIEW: WHAT CAN HAPPEN INSIDE TANKS?**

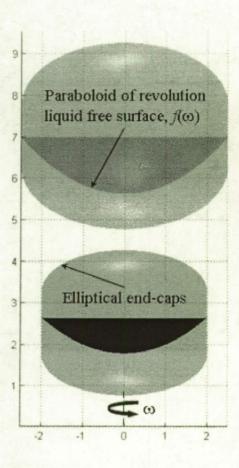
- Propellant T&P must be within specified range for turbomachinery operation
  - If propellants outside specified T&P box engine may not restart
  - Orbit cannot be circularized



## **MOTIVATION**

- Rotation present during missions to evenly heat spacecraft
- Effect rotation has on propellant thermal properties unknown
- Upgrade current analytical/numerical stratification models to include rotation





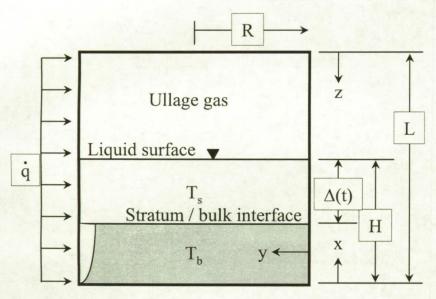
#### MISSION PARAMETER RANGES

- Tank Dimensions:
  - Square 3 m diameter tanks
- Cryogenics: LH<sub>2</sub>, LOX
  - T<sub>bulk</sub> LH<sub>2</sub>: 16 K, 28.8 °R, -430.9 °F
  - T<sub>bulk</sub> LOX: 91 K, 163.8 °R, -295.9 °F
- Tank Pressure (All Cryogenics): 30 psi
- Initial Fill Levels: 10, 20, 30%
- Heating Conditions:
  - Constant wall temperature:  $\theta = T_{\text{wall}} T_{\text{bulk}}$ :  $\Delta T = 0.1, 0.5, 1.0 \text{ K}$
  - Heat flux to fluid: 5-100 W/m<sup>2</sup>
- Reduced Gravity Environment:  $g/g_0 = 10^{-4}$ ,  $10^{-3}$ ,  $10^{-2}$ ,  $10^{-1}$ , 1
- Rotation rates:  $\omega = 0.1, 1, 5$  %ec
- Orbital Transfer Time (Simulation Time): 2-4 HR

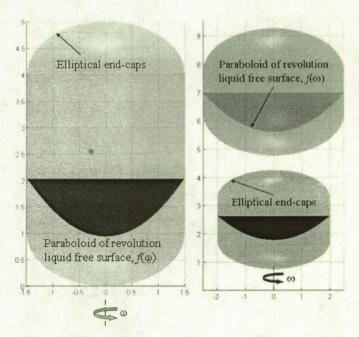
## **MASTER MODEL: BASIC FRONT END OPTIONS**

- 1. Tank geometry
  - Tank diameter, height
  - Square bottom
- 2. Boundary layer nature and heat transfer coefficient selection
  - Free convection
    - Laminar/Turbulent (w/ & w/out switching)
- 3. Wall temperature settings
  - Constant inner wall temperature
  - Constant inner wall heat flux
- 4. Rotation rate
- 5. Gravity level

#### GENERAL MODELING PHILOSOPHY



- Stratum growth  $\Delta(t)$ 
  - u(y) depends on if heating is constant wall temperature or constant heat flux, q
  - u(y) depends on nature of boundary layer
  - Provides differential equation for  $\Delta(t)$
- Stratum temperature,  $T_s(t)$ 
  - Heat entering side wall into boundary layer is used to increase stratum temperature
  - Energy exchange with ullage negligible
  - T<sub>s</sub> assumed uniform



$$\dot{m}_{bl} = 2\pi R \rho \int_{0}^{\delta} u(y) dy = \rho \pi R^{2} \frac{d\Delta}{dt}$$

$$\dot{q} 2\pi RH = \rho \pi R^2 \Delta c_p \frac{dT}{dt}$$

#### RELEVANT NON-DIMENSIONAL NUMBERS

$$Gr = \frac{g\beta\theta L^3}{v^2}$$

• Grashof number, **Gr**, governs heat transfer regime for constant wall temperature

Ratio of buoyancy to viscous forces

– β, Volumetric thermal expansion coefficient

- θ, Wall to Bulk temperature difference

$$Ra = Gr Pr$$

• Rayleigh number, **Ra**, is product of Grashof and usual Prandtl number, **Pr** 

• Prediction of boundary layer transition

- If  $\mathbf{Ra} < 10^9 \rightarrow \mathbf{Laminar}$ 

- If  $\mathbf{Ra} > 10^9 \rightarrow \text{Turbulent}$ 

$$Gr^* = \frac{g\beta q_w L^4}{kv^2}$$

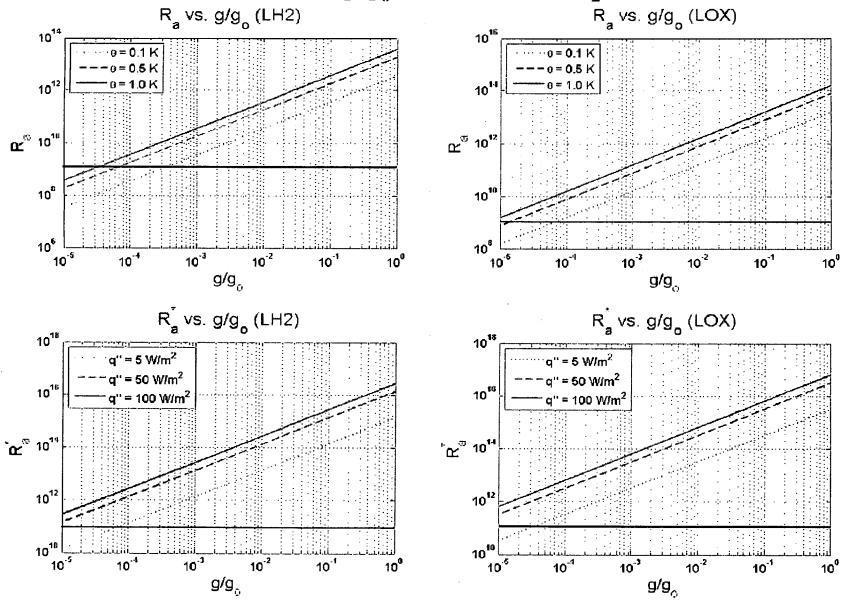
• Modified Grashoff number,  $\mathbf{Gr^*}$ , governs heat transfer regime for uniform heat flux,  $\mathbf{q_w}$ 

 $Ra^* = Gr^* \Pr$ 

• Modified Rayleigh number, Ra\*, for uniform heat flux

• Others: Reynolds Number, **Re** (momentum to viscous) Weber number, **We** (inertial to capillary), Froude number, **Fr** (inertial to body), and Bond number **Bo** (body to capillary)

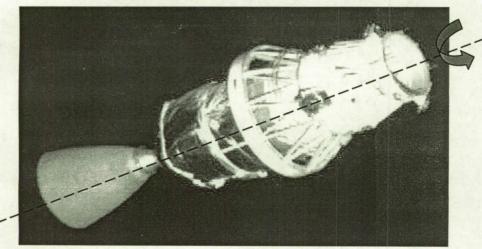
# Ra and Ra\* vs. g/g<sub>0</sub> MAPS for LH<sub>2</sub> and LOX



Maps laminar or turbulent boundary layers possible for typical mission profiles (NIST data)
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## WHAT IS IMPACT OF BBQ ROLL ROTATION:

Typical rotation rate, ω~1°/sec

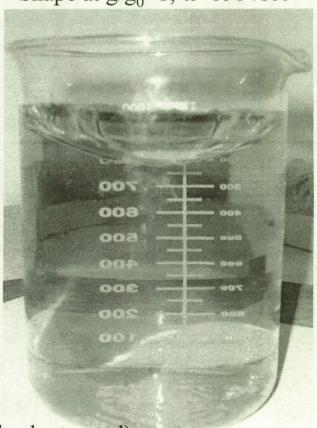


- Does 1º/sec matter?
- Not at  $g/g_0=1$  but in coast phase  $g/g_0\sim 10^{-4}$  $\rightarrow$  significant dishing effect

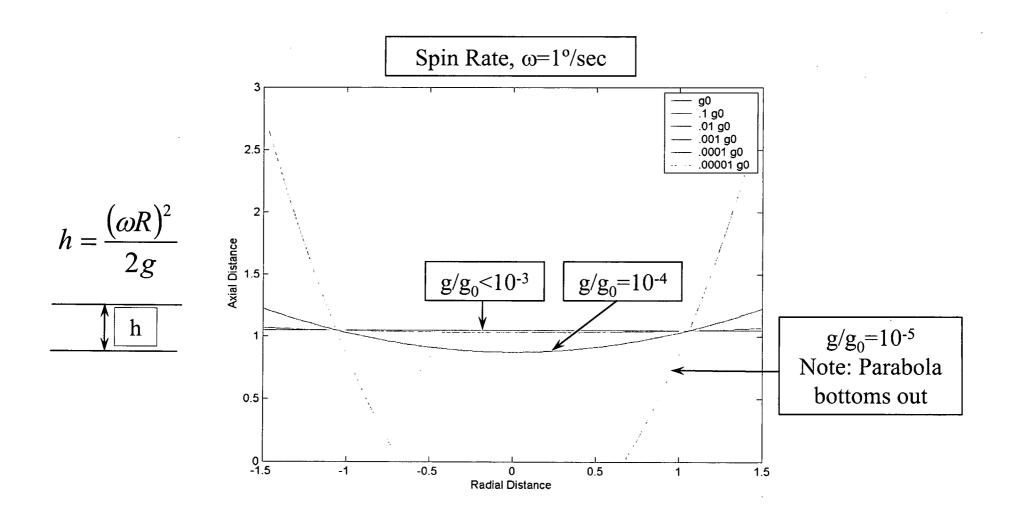
Key Question: How does rotation impact results?

- Assume liquid is in solid body rotation (transients can also be treated)
- · Model extra height that liquid gains along wall as a longer interfacial heat transfer length
- Center point in radial direction of tank is taken to be point where percent of bulk remaining is referenced → worst case scenario
- Trade off between heated area and surface area to distribute warm stratum

Shape at  $g/g_0=1$ ,  $\omega \sim 850^{\circ}/\text{sec}$ 



## **ROTATION / STRATIFICATION COMBINED MODEL**



#### **ROTATIONAL CASES**

Re-examine boundary layer / stratum mass balance

$$\dot{m}_{bl} = \pi R^2 \rho \left(\frac{d\Delta}{dt}\right) = S_{paraboloid} \rho \left(\frac{d\Delta}{dt}\right) = \frac{\pi R}{6h^2} \left[ \left(R^2 + 4h^2\right)^{3/2} - R^3 \right] \rho \left(\frac{d\Delta}{dt}\right)$$

Turbulent

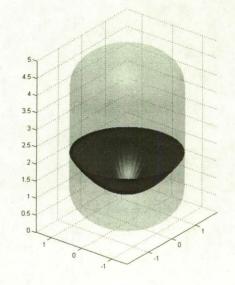
Laminar

$$\frac{\Delta(t)}{H_{\omega}} = 1 - \left[1 + 0.082 \frac{H_{\omega} \pi R}{S_p} \left(Gr_{\omega}^*\right)^{2/7} \frac{1}{\Pr^{2/7}} \phi_{\omega}\right]^{-7} \qquad \qquad \Delta(t) = 1 - \left[1 - 0.616 \frac{H_{\omega} \pi R}{S_p} \frac{\left(Gr_{\omega}^*\right)^{1/5}}{\left(\frac{4}{5} + \Pr\right)^{1/5}} \frac{1}{\Pr^{3/5}} \phi_{\omega}\right]$$

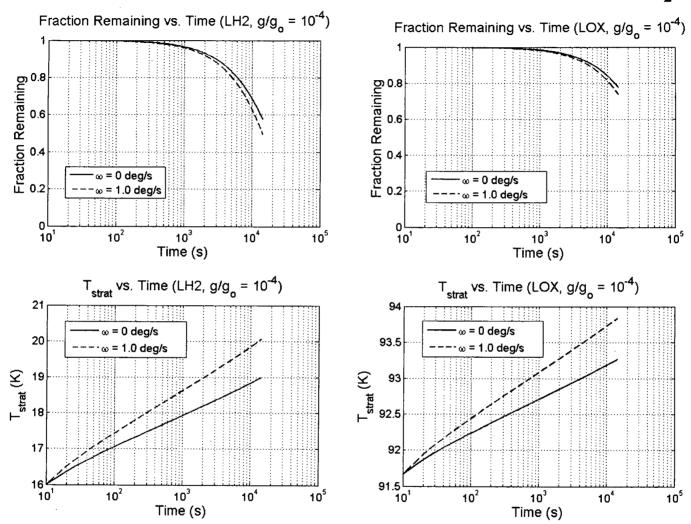
$$Gr^* = \frac{g\beta q_w \left(H + \frac{h}{2}\right)^4}{kv^2} = \frac{g\beta q_w (H_\omega)^4}{kv^2} \qquad \phi = \frac{vt}{H_\omega^2}$$

Re-derive energy balance to take into account additional heating area

$$\dot{q} 2\pi R \left( H + \frac{h}{2} \right) = \dot{q} 2\pi R H_{\omega} = \rho \pi R^2 \Delta c_p \frac{dT}{dt}$$



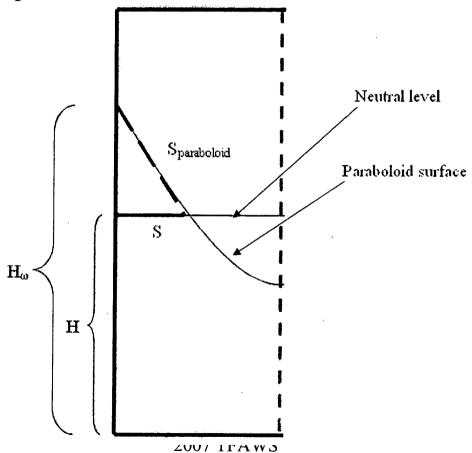
## COMBINED ROTATION / STRATIFICATION MODEL: LH<sub>2</sub> and LOX



- For q=10 W/m<sup>2</sup>, L=3, R=1.5, 20% fill level, H/R=0.4 and  $\omega$ =1°/sec at g/g<sub>0</sub>=10<sup>-4</sup>:
  - Rotation decreases time to stratification time by  $\sim 15\%$
  - Rotation increases stratification temperature by  $\sim 1.0 \text{ K}$

## **EFFECTS OF ROTATION AND TRADEOFFS**

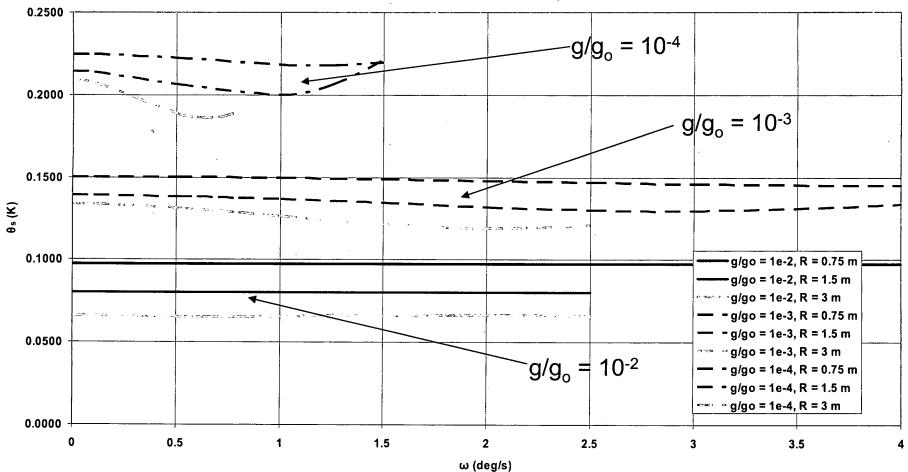
- Increased boundary layer running length  $(H \rightarrow H_{\omega})$ 
  - more heated area
  - larger Grashof number
- Larger surface area at bulk-stratum interface (S  $\rightarrow$  S<sub>paraboloid</sub>)
  - increased mass flow rate into stratum layer
  - more area to spread mass flow



#### **EFFECTS OF ROTATION**

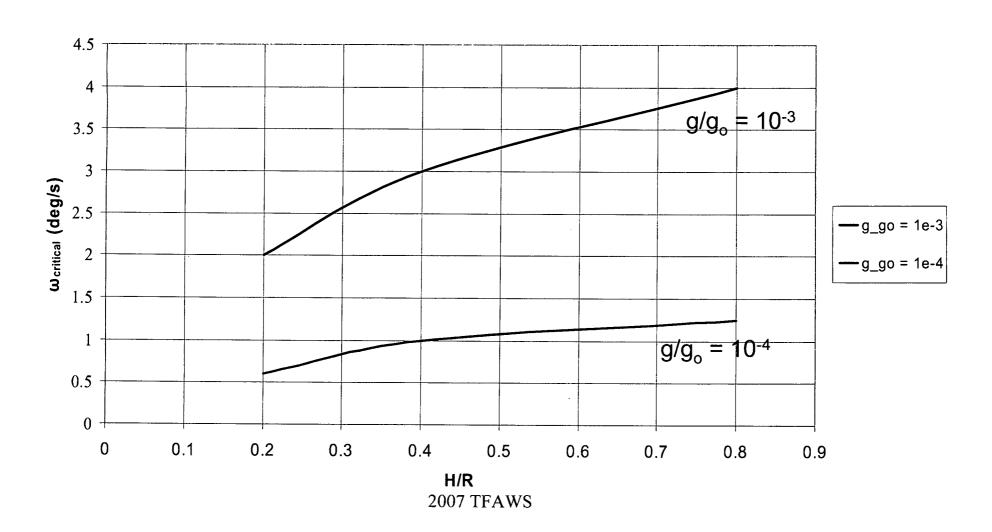
- Spinning always increases stratification
- Stratum temperature affected by spin rate; especially at low gravity levels
- LOX cases shown with heat flux of 5 W/m<sup>2</sup> after 2 hour mission





## **EFFECTS OF ROTATION**

- $\omega_{critical} \rightarrow$  spin rate to minimize stratum temperature
- $\omega_{critical}$  needed for large g/g<sub>o</sub> impractical
- $\omega_{critical}$  needed for typical mission profiles very practical ( $\omega < 1.5$  deg/s)
- LOX results discussed previously shown



## **SUMMARY/ CONCLUDING REMARKS**

- Thermal stratification impacts T&P at conclusion of coast phase
- Rotation (creeping of fluid up side walls) has large effect for  $\omega=1^{\circ}/s$  and  $g/g_0=10^{-4}$ 
  - 'Classical' literature model upgraded to include rotation effects
  - Can decrease time to stratify by 30-60 minutes during 4 hour coast
  - Larger heating area and lower liquid level above sump inlet
  - For various missions stratum temperature may increases or decrease relative to no-spin case
  - Mixed tank temperatures always larger because  $\Delta$  increased with rotation
- Future work
  - Comparison with CFD studies

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